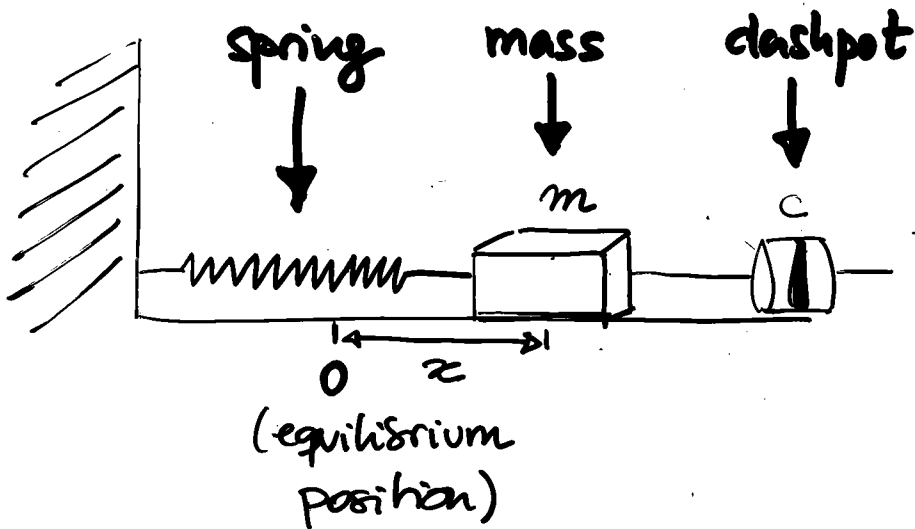


MECHANICAL VIBRATIONS



Force exerted by spring is $F_S(x) = -Kx$
($K > 0$ is the spring constant)

Force exerted by dashpot is $F_D(x) = -c \frac{dx}{dt}$
($c > 0$ is the damping constant)

Assume also that mass is subject to an external force $F_E = F(t)$.

Using Newton's law $F = ma$,

$$F(t) - Kx - c \frac{dx}{dt} = m \frac{d^2x}{dt^2}$$

or

$$m x'' + c x' + Kx = F(t)$$

SECOND-ORDER LINEAR DIFF EQUATION!

Free undamped motion

$$c=0 \text{ in } mx'' + cx' + kx = F(t)$$

$$F(t)=0$$

$$\text{That is, } mx'' + kx = 0.$$

Define $\omega_0 = \sqrt{\frac{k}{m}}$. Then we can write

$$x'' + \omega_0^2 x = 0$$

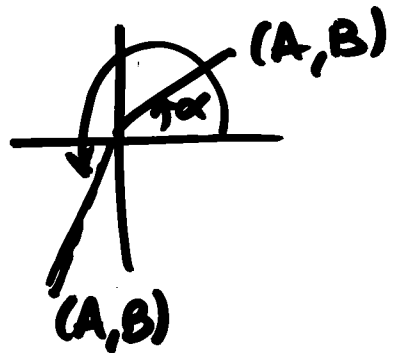
The general solution of this equation is of the form

$$x(t) = A \cos \omega_0 t + B \sin \omega_0 t$$

* Let's do a nice trick!

$$C = \sqrt{A^2 + B^2}$$

$$\alpha = \arctan(A, B)$$



Then

$$\cos \alpha = \frac{A}{C} \text{ and } \sin \alpha = \frac{B}{C}$$

We rewrite the solution $x(t)$ as

$$\begin{aligned} x(t) &= C \cdot \left(\frac{A}{C} \cos \omega_0 t + \frac{B}{C} \sin \omega_0 t \right) = \\ &= C \cdot \left(\cos \alpha \cos \omega_0 t + \sin \alpha \sin \omega_0 t \right) = \end{aligned}$$

$$x(t) = C \cdot \cos(\omega_0 t - \alpha)$$

Free damped motion

$$F(t) = 0 \text{ in } mx'' + cx' + kx = F(t)$$

$$\text{That is, } mx'' + cx' + kx = 0$$

Define $\omega_0 = \sqrt{\frac{k}{m}}$ and $p = \frac{c}{2m} > 0$. Then

$$x'' + 2px' + \omega_0^2 x = 0$$

The characteristic equation has roots

$$r^2 + 2pr + \omega_0^2 = 0$$

$$r = -p \pm \sqrt{p^2 - \omega_0^2}$$

- * If $p^2 - \omega_0^2 > 0$ (i.e., $c^2 > 4km$), then we have two distinct real roots r_1, r_2 , with $r_1, r_2 < 0$

$$x(t) = c_1 e^{r_1 t} + c_2 e^{r_2 t}$$

Therefore $x(t) \xrightarrow{t \rightarrow \infty} 0$

- * If $p^2 - \omega_0^2 = 0$ (i.e., $c^2 = 4km$), then we have a repeated real root $r = -p$, so

$$x(t) = e^{-pt} (c_1) + t e^{-pt} c_2$$

Again $x(t) \xrightarrow{t \rightarrow \infty} 0$

* If $p^2 - \omega_0^2 < 0$ (i.e., $c^2 < 4km$), then we have two distinct complex eigenvalues $-p \pm i\sqrt{\omega_0^2 - p^2}$. Then

$$x(t) = e^{-pt} (A \cos \omega_1 t + B \sin \omega_1 t)$$

where $\omega_1 = \sqrt{\omega_0^2 - p^2}$.

Using same trick as before, we can rewrite this as

$$x(t) = C e^{-pt} \cos(\omega_1 t - \alpha)$$

Again $x(t) \xrightarrow{t \rightarrow \infty} 0$

Undamped forced motion

$$m\ddot{x} + kx = F_0 \cos \omega t \quad \text{EXTERNAL FREQUENCY}$$

Solutions of associated homogeneous equation are

$$x = c_1 \cos \omega_0 t + c_2 \sin \omega_0 t, \quad \omega_0 = \sqrt{\frac{k}{m}}$$

NATURAL FREQUENCY

Assume $\omega \neq \omega_0$

We try $x_p = A \cos \omega t$ and substituting, we get

$$A = \frac{F_0}{k - m\omega^2} = \frac{F_0/m}{\omega_0^2 - \omega^2}$$

So general solution is

$$x(t) = c_1 \cos \omega_0 t + c_2 \sin \omega_0 t + \frac{F_0/m}{\omega_0^2 - \omega^2} \cos \omega t$$

RESONANCE

Note that when $\omega \rightarrow \omega_0$, $A \rightarrow \infty$

Let's solve the equation when $\omega = \omega_0$.

[Duplication: $\cos \omega t$ is a solution of the associated homogeneous equation].

We try $x_p = t(A \cos \omega_0 t + B \sin \omega_0 t)$, yielding

$$x_p(t) = \frac{F_0}{2m\omega_0} t \sin \omega_0 t$$

GROWS
UNBOUNDED!

